# Effect of the Dowel Length, Dowel Diameter, and Adhesive Consumption on Bending Moment Capacity of Heat-treated Wood Dowel Joints

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This study applied response surface methodology for modeling and optimizing heat-treated wood dowel joints, the most used joint in furniture construction. The factors examined were dowel length, dowel diameter, and adhesive consumption. The bending moment capacity of the joints loaded in compression or tension were the responses. The load was applied at a constant speed until a major separation between the two parts occurred. To figure out the bending moment capacity, the ultimate failure loads and the moment arms were obtained during testing the joints. The joints were tested by using a universal testing machine. A two-factor interaction model was established to describe the relationship between the factors and the responses. An analysis of variance was employed to test the significance of the developed mathematical model. The dowel length, dowel diameter, and adhesive consumption had significant effects on the bending moment capacity of the heat-treated dowel joints. The dowel length was the main factor that affected the bending moment capacity of the heat-treated dowel joints.

*Keywords: Heat-treated wood; Wood joints; Tension and compression loading; Bending moment capacity Response surface methodology; Optimization* 

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#### INTRODUCTION

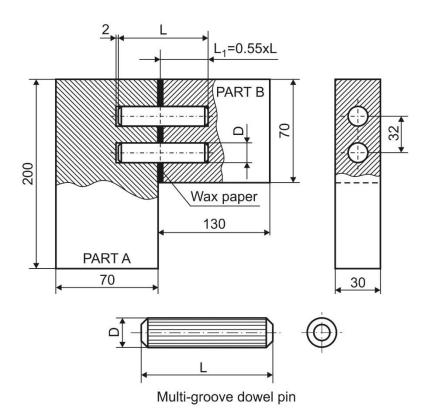
Heat treatment is an effective method to enhance wood dimensional stability. It has been developed to enhance wood properties and make wood competitive against other materials (Dilik and Hiziroglu 2012; Sandberg *et al.* 2013). Outdoor furniture is one potential application of heat-treated wood (Sandberg *et al.* 2013; Kuzman *et al.* 2015).

The furniture structural system can be classified as frame- or case-type construction (Smardzewski 2015). When a combination of these solutions are observed in the same product, the designed furniture has a composite structure (Eckelman 2003). However, the designed structure must resist to the loads that are going to act on the furniture.

The most used joint in furniture construction is the dowel joint (Fig. 1) because it has favorable cost and production characteristics (Eckelman 2003; Diler *et al.* 2017). Various factors can affect the strength of dowel joints, including wood species, dowel length, depth of dowel embedment, dowel type, hole diameter, distance between holes, number of dowels, adhesive type and consumption, tightness of fit, boring speed, and feed rate (Eckelman 2003).

Dowel joint sizing (*e.g.*, dowel length, dowel diameter, and adhesive consumption) is based on the studies that have been developed for untreated wood (Curtu *et al.* 1988; Eckelman 2003; Cismaru 2009; Smardzewski 2015). Because heat-treated wood has

inferior mechanical properties compared to untreated wood, an optimization tool could be used in the designing phase of the furniture for appropriate sizing of joints made of heat-treated wood (Tankut *et al.* 2014).



**Fig. 1.** The aspect and main characteristics of the analyzed L-shaped dowel joint (dimensions in mm) (Craftsmanspace 2019)

Moreover, there is limited information on the influence of various factors on the strength of heat-treated wood dowel joints. Kuzman *et al.* (2015) studied the effect of heat treatment on the mechanical properties of dowel joints produced from untreated and heat-treated beech (*Fagus sylvatica* L.) and spruce (*Picea abies* L.). Diler *et al.* (2017) determined that wood species, heat treatment, adhesive, and joint type have significant effects on the withdrawal force capacity of dowel joints. Georgescu and Bedelean (2017) analyzed the effect of heat treatment, dowel length, dowel spacing, and depth of dowel embedment on the mechanical strength of dowel joints made of ash (*Fraxinus excelsior*). Their results matched Kuzman *et al.* (2015): The joints of untreated wood had lower compressive and tensile failure loads than the joints of untreated wood. Additionally, Georgescu and Bedelean (2017) reported that the compressive and tensile strengths of heat-treated dowel joints increases when the dowel length increases, the distance between holes increases, and the ratio of dowel embedment in the rail of the joint decreases. However, the analyzed factors could not fully describe the behavior of heat-treated wood dowel joints, and other factors must be considered.

Therefore, the objective of this study was to examine the influence of dowel diameter, adhesive consumption, and dowel length on the bending moment capacity of heat-treated wood dowel joints using response surface methodology (RSM). Design and analysis with RSM consist of designing and executing an experiment to generate response

data, fitting the data to a series of polynomial models, conducting an analysis of variance to assess factors' statistical significance, comparing models, selecting the simplest model that best predicts the analyzed response, and using the selected model to determine the influence of the factors and to reveal the optimal configuration (Anderson and Whitcomb 2005; Yuan *et al.* 2015).

### EXPERIMENTAL

#### Materials

The wood used in this study, heat-treated ash (*Fraxinus excelsior*) boards, was obtained from a local sawmill located in Brasov, Romania. Some basic technological steps (namely, straightening, planing, ripping, and crosscutting at final dimensions) were followed to obtain the parts of the dowel joint (parts A ( $200 \times 70 \times 30 \text{ mm}$ ) and B ( $130 \times 70 \times 30 \text{ mm}$ )) (Fig. 1). Prior to boring, the parts were visually sorted based on several criteria, namely, lack of structural and technological defects and presence of radial annual ring orientation, to maximize the joint reliability (Záborský *et al.* 2017).

The sorted parts (682 pieces for part A and 703 pieces for part B) were weighed using a digital scale and grouped (based on their weight) into classes using frequency analysis techniques. Parts of the joints were randomly selected from each class, to assure that all groups were similar in terms of material characteristics.

Polyvinyl acetate adhesive (Kleiberit 303 (D4); Kleiberit, Weingarten, Germany) was uniformly dosed and applied in each hole of the joints using a syringe and a glass rod. The quantity of adhesive was calculated by multiplying the adhesive consumption rate by the area of each hole. The possible influence of excess adhesive on the strength of the joints was limited using wax paper (Fig. 1) to separate the parts of the joints (Dalvand *et al.* 2014).

Multi-grove dowel pins (Fig. 1) made of beech wood (*Fagus sylvatica*) was used to assemble the parts of the joints. The depth of dowel penetrations in the rail (part B of the joint) was 0.55 from its length (Fig. 1).

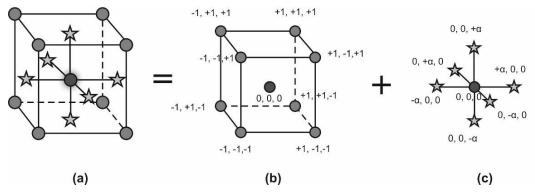
The steps to assemble one joint were: applying the adhesive on each wall of the hole; inserting the dowels in the rail; applying the wax paper; mounting the other part of joint; and pressing the joint in a wood clamp until 24 h.

#### Methods

To obtain the configurations of dowel joints, a central composite design (CCD) was generated with the statistical package Design-Expert<sup>®</sup> (version 9, Stat-Ease Inc., Minneapolis, MN, USA). In this experimental design, the factors were dowel length ( $X_1$ ), dowel diameter ( $X_2$ ), and adhesive consumption ( $X_3$ ). The responses were bending moment capacities of joints loaded in compression ( $\hat{Y}_{MC}$ ) or tension ( $\hat{Y}_{MT}$ ). The CCD was built up from the following (Anderson and Whitcomb 2005; Ariaee *et al.* 2014):

• Two-level factorial design points (Fig. 2b) (estimating first-order and twofactor interactions). These points take into account all possible combination of the low (-1) and high (+1) levels of analyzed factors. In this study, there were three factors and each analyzed at two levels (-1, +1). Therefore, there were eight possible combinations (+1, +1, +1), (+1, +1, -1), (+1, -1, -1), (-1,-1,-1), (-1, +1,+1), (-1, +1, +1), (-1, +1, -1) and (+1, -1, +1). Based on these combinations, the configurations #1, 2, 4, 5, 6,11, 12 and 14 of the joints were constructed (Table 1);

- Axial points (estimating pure quadratic effects). Two factors were analyzed at the center value (0) and the third one has the value of alpha (α) (Fig. 2c). Therefore, in this study, there were six possible combinations, namely, (-α, 0, 0), (+α, 0, 0), (0, 0, +α), (0, 0, -α), (0, +α, 0) and (0, -α, 0). Based on these combinations the configurations #3, 7, 8, 9, 10 and 15 of the joints were constructed (Table 1);
- Center points (estimating the experimental error). Each factor was analyzed at the center value (Fig.2b), namely, the configuration #13 (Table 1) of analyzed joints was constructed.



**Fig. 2.** Face centered experimental design (a), two level factorial design and center points (b) and axial points (c) (Anderson and Whitcomb 2005)

**Table 1.** Factors and Corresponding Levels for the Applied Experimental Design

Factor	Level							
	-α*	-1	0	+1	+α*			
Dowel length $(X_1)$ (mm)	30	30	50	70	70			
Dowel diameter (X <sub>2</sub> ) (mm)	6	6	8	10	10			
Adhesive consumption (X <sub>3</sub> ) (g/m <sup>2</sup> )	250	250	350	450	450			
* For the applied design, namely, the face centered design, $\alpha = 1$ (Anderson and Whitcomb 2005)								

The analyzed configurations are presented in Table 2. Each configuration was replicated 15 times. Therefore, a total of 450 joints were assembled and tested. Following assembly, the joints were conditioned for at least one month in the same area where the compressive and tensile tests were performed (Kasal *et al.* 2015). The equilibrium moisture content of heat-treated wood joints was about 5%, and the average density was equal to 618 kg/m<sup>3</sup>. Half of the joints were subjected to compression tests, and the other half were subjected to tension tests.

The mechanical testing of joints was performed on a universal testing machine (Zwick Roell Z10; Zwick GmbH & Co. KG, Ulm, Germany). The load was applied at a constant speed of 3 mm/min until a notable separation between the two parts occurred (Kuzman *et al.* 2015). The ultimate failure load was recorded for each analyzed dowel joint. The compression and tension tests simulated two possible situations, which could be observed when a force is applied to a typical frame structure. In the first situation, the

corner joint was subjected to a compression load (Fig. 3a). In the other, the corner joint was subjected to a tension force (Fig. 3b) causing a moment tending to open the joint (Negreanu 2003; Yerlikaya and Aktaş 2012; Kasal *et al.* 2015).

<b>Table 2.</b> Configuration of Joints According to the Experimental Plan and the
Mean Experimental Values with Coefficient of Variation

		Factors	Responses						
Configuration	Dowel	Dowel	Adhesive	Bend	ling momen	t capaci	t capacity (Nm)		
No.	Length	Diameter	Consumption	Compression		Tension			
	(X <sub>1</sub> )	(X <sub>2</sub> )	(X <sub>3</sub> )	. (	Y <sub>1</sub> )	(Y <sub>2</sub> )			
	(mm)	(mm)	(g/m²)						
				M CV (%)		М	CV (%)		
1	70 (+1)	6 (-1)	450 (+1)	135	18	228	14		
2	30 (-1)	10 (+1)	250 (-1)	68	9	129	11		
3	50 (0)	8 (0)	450 (+1)	124	13	221	16		
4	70 (+1)	10 (+1)	250 (-1)	142	10	241	14		
5	30 (-1)	6 (-1)	450 (+1)	58	22	97	26		
6	30 (-1)	10 (+1)	450 (+1)	103	17	198	11		
7	70 (+1)	8 (0)	350 (0)	139	15	258	19		
8	30 (-1)	8 (0)	350 (0)	72	18	137	17		
9	50 (0)	6 (-1)	350 (0)	81	18	134	15		
10	50 (0)	8 (0)	250 (-1)	93	22	160	13		
11	70 (+1)	10 (+1)	450 (+1)	196	12	399	12		
12	70 (+1)	6 (-1)	250 (-1)	95	14	152	21		
13	50 (0)	8 (0)	350 (0)	106	15	204	15		
14	30 (-1)	6 (-1)	250 (-1)	43	23	85	17		
15	50 (0)	10 (+1)	350 (0)	130	9	237	16		
M – mean, CV – coefficient of variation; the coded values of the experimental plan are									
presented in parentheses									

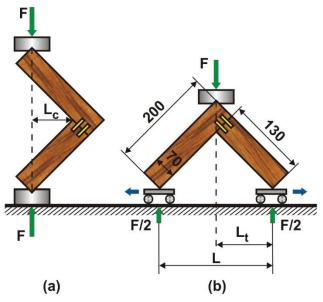


Fig. 3. The loading forms of dowel joints subjected to compression (a) and tension (b) (dimensions in mm)

The bending moment capacity of the analyzed joints was calculated by means of Eq.1 for compression test and Eq. 2 for tension test (Derikvand and Eckelman 2015),

$$\boldsymbol{M}_{\boldsymbol{c}} = \boldsymbol{F} \times \boldsymbol{L}_{\boldsymbol{c}} \tag{1}$$

$$\boldsymbol{M}_{t} = \frac{F}{2} \times \boldsymbol{L}_{t} \tag{2}$$

where  $M_c$  is the bending moment of joints subjected to compression (Nm),  $M_t$  is the bending moment of joints loaded in tension, F is the ultimate failure load (N),  $L_c$  is compression moment arm (42 mm), and  $L_t$  is the tension moment arm (92 mm).

#### **RESULTS AND DISCUSSION**

The bending moment capacities of joints that were under a diagonal tension load were, on average, about 45% higher than those of the joints loaded in compression (Table 2). The coefficient of variation was between 9 and 23% for compression sets and between 11 and 26% for the joints loaded in tension. Therefore, the bending moment capacity in tension was more variable than that in compression.

In general, the failure modes of joints included glue-line fracture, dowel deformation, and dowel and material fracture (Fig. 4). Displacement and deformation of the dowels was also observed. Also, a separation of the parts of the joints was frequently observed when the dowel length, dowel diameter and adhesive consumption were tested at the minimum value (30 mm, 6 mm and 250 g/m<sup>2</sup>). On the other hand, when the dowel length, dowel diameter analyzed at maximum value (70 mm, 10 mm and 450 g/m<sup>2</sup>), either material fracture or dowel fracture was observed. These failures occurred both during compression and tension testing.

Two-factor interaction (2FI) empirical models were suggested by the statistical software to predict the bending moment capacity under diagonal compression and tension loading of heat-treated wood dowel joints. The models were significant at the 1% level. The mathematical models that could predict the bending moment capacity of the joints loaded in compression ( $\hat{Y}_{Mc}$ ) or tension ( $\hat{Y}_{Mt}$ ) are presented in terms of both coded and actual factors (Eqs. 3 to 6):

$$\hat{Y}_{\text{Mc, coded}} = 105.75 + 36.31X_1 + 22.69X_2 + 17.55X_3 + 4.60X_1X_2 + 5.34X_1X_3 + 4.17X_2X_3$$
(3)

 $\hat{Y}_{\text{Mc, actual}} = 13.919 - 0.039X_1 - 1.705X_2 - 0.124X_3 + 0.115X_1X_2 + 0.002X_1X_3 + 0.020X_2X_3$ (4)

 $\hat{Y}_{\text{Mt, coded}} = 192.06 + 63.18X_1 + 50.73X_2 + 37.64X_3 + 14.36X_1X_2 + 19.08X_1X_3 + 17.26X_2X_3$ (5)

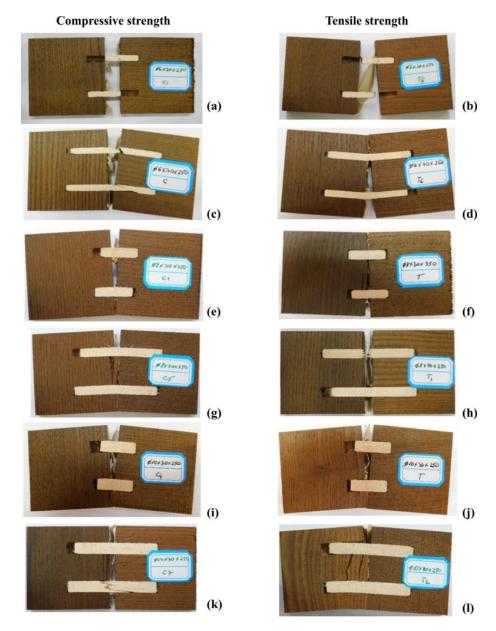
 $\hat{Y}_{\text{Mt, actual}} = 251.56 - 3.050X_1 - 22.784X_2 - 0.790X_3 + 0.358X_1X_2 + 0.009X_1X_3 + 0.086X_2X_3$ (6)

The coded equations (Eqs. 3 and 5) are useful for identifying the relative impacts of the factors (Sova *et al.* 2016). Meanwhile, the equations in terms of actual factors (Eqs. 4 and 6) can be used to make predictions about the bending moment capacity of the heat-

treated dowel joints. The relation between the coded and actual values is described by the Eq. 7 (Zhu *et al.* 2015),

$$X_i = \frac{(A_i - A_0)}{\Delta A} \tag{7}$$

where  $X_i$  is the coded value of the analyzed factor,  $A_i$  is the actual value of the analyzed factor,  $A_0$  is the actual value at the center point of the factor (Table 1),  $\Delta A$  is the step change.



**Fig. 4.** Failure modes of the analyzed dowel joints, including glue-line fracture (a, b, and e), material fracture (f, g, k, and l), dowel fracture (c and h), and deformation and displacement (d, j, and i)

**Table 3.** Analysis of Variance Results for the 2FI Equation of the StatisticalSoftware for the Bending Moment Capacity of the Joints Loaded in Compression

"Source"	"Sum of Squares"	Sumoi <sub>"df</sub> " iviean F-		"p-value Prob > F"	Observation	
Model	378000	6	63002	213.68	< 0.0001	
Dowel length $(X_1)$	227100	1	227100	770.14	< 0.0001	
Dowel diameter (X <sub>2</sub> )	88689	1	88689	300.80	< 0.0001	Significant
Adhesive consumption (X <sub>3</sub> )	53009	1	53009	179.79	< 0.0001	
$X_1X_2$ interaction	2920	1	2920	9.90	0.0019	
$X_1X_3$ interaction	3927	1	3927	13.32	0.0003	
$X_2X_3$ interaction	2395	1	2395	8.13	0.0048	
Residual	64276	218	294			
Lack of fit	917	8	114	0.38	0.9304	Not significant
Pure error	63359.63	210	301			
Corrected total	442300	224				
Predicted R <sup>2</sup>	0.845					
Adjusted R <sup>2</sup>	0.850					

**Table 4.** Analysis of Variance Results for the 2FI Equation of the Statistical

 Software for the Bending Moment Capacity of the Joints Loaded in Tension

"Source"	"Sum of Squares"	"df"	"Mean Square"	"F- value"	"p-value Prob > F"	Observation
Model	1246000	6	207600	229.06	< 0.0001	
Dowel length $(X_1)$	573100	1	573100	632.27	< 0.0001	
Dowel diameter (X <sub>2</sub> )	369500	1	369500	407.68	< 0.0001	
Adhesive consumption (X <sub>3</sub> )	203400	1	203400	224.43	< 0.0001	Significant
$X_1 X_2$ interaction	23667	1	23667	26.11	< 0.0001	
$X_1X_3$ interaction	41800	1	41800	46.12	< 0.0001	
$X_2X_3$ interaction	34217	1	34217	37.75	< 0.0001	
Residual	197600	218	906			
Lack of fit	7474	8	934	1.03	0.4130	Not significant
Pure error	190100	210	905			
Corrected total	1443000	224				
Predicted R <sup>2</sup>	0.854					
Adjusted R <sup>2</sup>	0.859					

The selected models obtained greater adjusted and predicted  $R^2$  (Tables 3 and 4) values in comparison with other models (linear, quadratic, and cubic). The predicted  $R^2$  was in reasonable agreement with the adjusted  $R^2$ . Moreover, the lack of fit was not significant. Therefore, the models adequately described the analyzed responses (Anderson and Whitcomb 2005). According to the ANOVA results (Tables 3 and 4), all main factors ( $X_1, X_2$ , and  $X_3$ ) and their interactions ( $X_1X_2, X_1X_3$ , and  $X_2X_3$ ,) were statistically significant at the 5% level.

Based on the obtained models (Eqs. 3 and 5) the main factor that affected the bending moment capacity of heat-treated wood dowel joints loaded in compression or tension was the dowel length ( $X_1$ ), followed by dowel diameter ( $X_2$ ) and adhesive consumption ( $X_3$ ). All three factors had synergetic (interaction) effects in increasing the bending moment capacity under diagonal compression and tension loading of the heat-treated wood dowel joints. The relative magnitudes of these interactions were in the order of  $X_1X_3 > X_1X_2 > X_2X_3$  for the diagonal compression loading and  $X_1X_3 > X_2X_3 > X_1X_2$  for the diagonal tension loading of the heat-treated wood dowel joints.

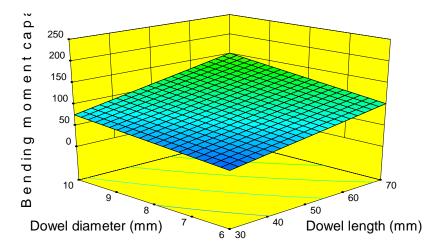
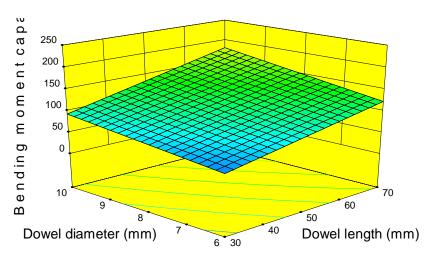
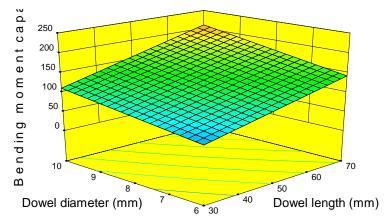


Fig. 5. 3D plot showing the interaction effects of dowel length and dowel diameter on the bending moment capacity of joints loaded in compression when the adhesive consumption was set at 250  $g/m^2$ 

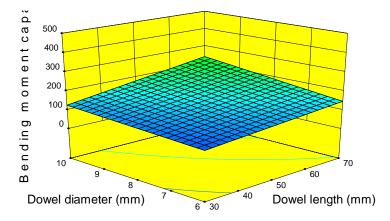


**Fig. 6.** 3D plot showing interaction effects of dowel length and diameter on bending moment capacity of joints loaded in compression when adhesive consumption was set at 350 g/m<sup>2</sup>

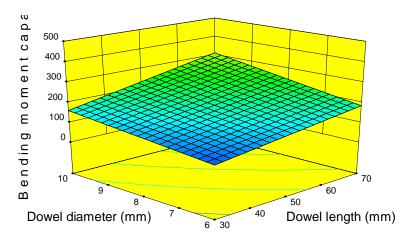
The interaction effects of dowel length and dowel diameter on the bending moment capacity of joints loaded in compression or tension are shown in Figs. 5 through 10. The greatest bending moment capacity can be achieved by increasing the dowel length, dowel diameter, and adhesive consumption (Figs. 7 and 10).



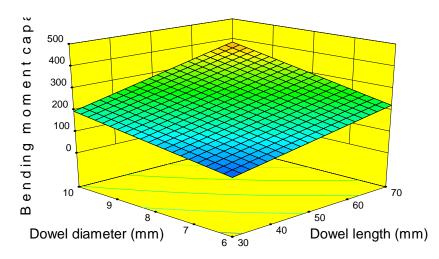
**Fig. 7.** 3D plot showing interaction effects of dowel length and diameter on the bending moment capacity of joints loaded in compression when the adhesive consumption was set at 450 g/m<sup>2</sup>

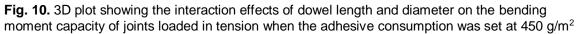


**Fig. 8.** 3D plot showing the interaction effects of dowel length and dowel diameter on the bending moment capacity of joints loaded in tension when the adhesive consumption was set at 250 g/m<sup>2</sup>



**Fig. 9.** 3D plot showing the interaction effects of dowel length and dowel diameter on the bending moment capacity of joints loaded in tension when the adhesive consumption was set at 350  $g/m^2$ 





The analyzed factors were optimized based on three scenarios. In the first optimization scenario, it was supposed that in the design phase of the product, one wants to achieve the maximum bending moment capacity of the heat-treated wood dowel joints. The second optimization scenario took into account the target value of 2225 N, which is the upper level of the vertical seat loads that can be transferred on the structure of a chair (Eckelman 2003). The third optimization scenario sought to achieve the target value of 1780 N, which is the upper level of the vertical loads that can be transferred on the arms of a chair (Eckelman 2003).

The analyzed factors were kept within the range of the data as given by the CCD experimental plan (Table 2). Equal importance was assigned to all of the analyzed factors (Table 5). The numerical optimization algorithm included in the statistical software was run to evaluate the optimal values as presented in Table 5. The solution with the highest desirability coefficient (D) was selected for each analyzed scenario. The relative error was computed using Eq. 8,

$$E_R = \frac{|Y - \hat{Y}|}{Y} \times 100 \tag{8}$$

where  $E_{\rm R}$  represents the relative error (%), Y is the experimental value (N), and  $\hat{Y}$  is the predicted value (N).

Factor / Response	Goal	Lower Limit	Upper Limit	Importance
Dowel length $(X_1)$ (mm)	In range	30	70	3
Dowel diameter (X <sub>2</sub> ) (mm)	In range	6	10	3
Adhesive consumption ( $X_3$ ) (g/m <sup>2</sup> )	In range	250	450	3
Bending moment capacity of joints loaded in compression (Y <sub>C</sub> ) (Nm)	Maximize	24	242	3
Bending moment capacity of joints loaded in tension (Y <sub>T</sub> ) (Nm)	Maximize	53	437	3

**Table 5.** Criteria for Different Factors and Responses in Optimization of Heattreated Wood Dowel Joints
 **Table 6.** Experimental Summary of Optimization of Heat-treated Wood Dowel

 Joints

				Bending moment capacity (Nm)						
Scenario	<b>X</b> 1	<b>X</b> <sub>2</sub>	X <sub>3</sub> Compression Tension			Compression				
No.	(mm)	(mm)	(g/m²)							D
				Ŷc	Yc	<i>E</i> <sub>R</sub> (%)	Ŷτ	Υ <sub>T</sub>	<i>E</i> <sub>R</sub> (%)	
1	70	10	450	210	168	25	386	298	29	0.86
2	70	6	250	103	83	24	147	141	5	0.89
3	40	6	420	80	67	19	122	132	8	0.92

One can observe that the relative error  $(E_R)$  was between 5% and 29% in predicting the bending moment capacity of heat-treated wood dowel joints (Table 6). In the case of the first scenario, namely, when the maximum bending moment capacity was planned to be achieved, the relative error was 25% in case of joints loaded in compression and 29% for the joints loaded in tension. For the second analyzed scenario, when the target value of 2225 N was imposed during optimization study, the relative error was 24% for compression and 5% for tension models. On the other hand, when the target value was 1780 N, the relative error was 19% for the compression test and 8% for the tension test. The obtained errors could be considered reasonable. Therefore, the proposed methodology could represent a tool for predicting and optimizing the bending moment capacity of heat-treated wood dowel joints loaded in compression or tension.

## CONCLUSIONS

The effect of dowel length, dowel diameter, and adhesive consumption on the bending moment capacity of heat-treated wood dowel joints loaded in compression or tension was analyzed in this study. Moreover, two regression equations were developed to predict the bending moment capacity of heat-treated wood dowel joints based on the analyzed factors.

- 1. Dowel length, dowel diameter, and adhesive consumption had significant effects on the bending moment capacity of heat-treated wood dowel joints.
- 2. The dowel length was the main factor affecting the bending moment capacity under diagonal tension and compression loading of the heat-treated dowel joints.
- 3. The bending moment capacity of joints loaded in tension was, on average, about 45% higher than that of the joints loaded in compression.
- 4. The optimum dowel length, dowel diameter, and adhesive consumption were revealed for the analyzed scenarios using response surface methodology.

## ACKNOWLEDGMENTS

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