# On Acoustic Emission Analysis in Circular Saw Cutting Beech Wood with Respect to Power Consumption and Surface Roughness

Srdjan Svrzic,<sup>a</sup> Marija Djurkovic,<sup>a,\*</sup> Gradimir Danon,<sup>a</sup> Mladen Furtula,<sup>a</sup> and Damjan Stanojevic<sup>b</sup>

A sound or a noise that accompanies wood machining processes is introduced by the tool rotation itself, by the friction of moving machine parts, or by wood-tool interaction. The sounds generated during machining with a circular saw could be analysed in order to monitor and possibly control the cutting process. Applying altered cutting parameters while cutting beech wood (Fagus sylvatica L.), which is the most common wood species in the Republic of Serbia, caused acoustic emissions that could be analysed throughout corresponding spectra. As shown in previous studies, altering the cutting parameters, e.g., the feed speed and tool override, resulted in variations in power consumption, surface roughness, and acoustic emission (or acoustic pressure). The aim of this paper was to provide a possible correlation between the applied cutting parameters and the acoustic emission spectra with respect to consumed power and the state of the machined surface. Along with acoustic emissions, the power consumption and surface roughness data were also acquired in order to make a possible relationship. By associating the idle circular saw acoustic spectra with background noise and comparing them with those obtained during machining, it was possible to indicate spectrum areas of particular interest for further analysis.

Keywords: Acoustic emission; Acoustic pressure; Wood machining; Cutting power; Surface roughness Circular saw; Spectrum analysis

Contact information: a: Department of Wood Science and Technology, University of Belgrade Faculty of Forestry, Belgrade 11000 Serbia; b: Department of Furniture and Interior Engineering, Academy of Technical and Educational Vocational Studies, Niš 18000 Serbia; \* Corresponding author: marija.djurkovic@sfb.bg.ac.rs

### INTRODUCTION

Different methods for monitoring wood machining processes are used for the purpose of obtaining proper work piece quality. A work piece should have the specified dimensions and desired surface characteristics. Along with the aforementioned results, the energy consumption, operational time, and machining system gearing state are observed as influential factors for desired machining process performance. The power consumption expressed by cutting power, along with acoustic emission, could potentially provide satisfactory data for permanent process monitoring (Mandić *et al.* 2015; Porankiewicz *et al.* 2021).

The acoustic emissions present an inevitable wood machining output. This comes from the moving (predominately rotating) parts of a machining system, producing vibrations and oscillatory sounds, which could be associated with the working state of the machine. Furthermore, the tool rotation and its self-generated and induced vibrations produce acoustic emission correlated to the state of the tool and its interaction with the work piece, *i.e.*, wood or wood based, material. An idling noise is produced from different sources: the air vortexes behind a tooth and the natural frequency of the saw blade causing lateral moving of the tool (Hattori and Iida 1999; Nasir and Cool 2020). The idling noise does not completely disappear during the cutting process, although the vortexes are much smaller due to the chips. In addition, the lateral movement is less as a result of saw fixation inside the material.

An adaptive control system based upon acoustic emissions in terms of surface roughness by adjusting the feed rate was proposed by Deja and Licow (2020) and monitoring sawing processes *via* means of acoustic emission was proposed by Nasir *et al.* (2019). Sound techniques were also proposed for cutting conditions (Nagatomi *et el.* 1993) and tool weariness monitoring (Banshoya *et al.* 1994).

Contact vibration measurement sensors, *e.g.*, accelerometers, are difficult to apply because they need to be placed as close as possible to the tool support (Delio *et al.* 1992). Contactless indirect vibration measurements providing acoustic emission spectra could be provided using a microphone as a sensor. The primary problem when using microphone is the inevitable mixing of sounds originating from the background, the machining system itself, or other sources. This was the primary reason for the purpose of this study; first, the idling acoustic spectra were recorded, and then these were compared with the acoustic spectra when the tool was interacting with the wood material.

Acoustic emission analysis, or acoustic pressure (AP), are currently based upon the determination of the natural frequencies of the cutting tools (predominantly circular saw blades), the critical rotation speeds introducing resonance (Orlowski *et al.* 2007), and the stability, lateral stiffness, levelling, and tensioning (Stakhiev 2003). Examinations were also directed towards the lowering idle noise of circular saw blades (Hattori and Lida 1999; Beljo-Lučić and Goglia 2001; Kopecky and Rousek 2012). Finally, the most important areas of implementation of the sound frequency analysis, from the perspective of this paper, were used as an indirect indicator of the tool wear (Suetsugu *et al.* 2005; Wilkowski and Górski 2011), wood surface roughness, wood fiber direction, feed rate, and the cutting width during routing (Iskra and Tanaka 2005, 2006; Durcan and Burdurlu 2018).

Parallel to recording the AP spectra, the power consumption was also measured. The power consumption measuring method was the AC voltage drop on the drive electromotor measured with a three-phase watt-meter.

Since the feed rate has a large impact upon the cutting power and the tool override influence on the circular saw lateral movement, it was reasonably to expect them to have mutual effects on the acoustic emission recordings obtained during the tool-wood interaction.

The hypothesis of this study was that different values of the feed rate and the tool override would elevate the AP at the certain frequencies of the acoustic emission spectra in a predictable manner, and that output can provide new possibilities for monitoring the cutting process in the terms of the power consumption and the surface quality for the circular saw cutting of beech wood.

### EXPERIMENTAL

In this study, beech (*Fagus sylvatica* L.) wood, as a widely spread tree species in the Republic of Serbia, was chosen for the material for this experiment. The average moisture of the beech planks was 9.04%, with a density of approximately 0.680 g/cm<sup>3</sup>. The dimensions of the planks were 1000 mm × 150 mm × 30 mm. The specimens were later cut along the grain. The samples had been conditioned before testing in the laboratory environment conditions: relative humidity of  $45 \pm 5\%$  and room temperature of  $20 \pm 3^{\circ}$ C.

The research presented in this paper was conducted at the Laboratory for Machines and Apparatus at the Faculty of Forestry, University of Belgrade (Beograd, Serbia). The machining system used for cutting the wood samples was a Minimax CU 410K combined machine (SCM, Rimini, Italy) equipped with a 3 kW three-phase asynchronous electrical motor. The exhausting system was involved throughout the entire experiment. The feed rate measurement took place before each series of samples *via* a Maggi Engineering Vario feed device (Maggi Technology, Certado, Italy), equipped with a 0.45 kW three-phase asynchronous electrical motor with an available measuring range of feed rates from 3 m/min to 24 m/min, which covers most common feed rates in practice. The tool overrides were set by means of a machine corresponding mechanical system deploying spindle and gears.

The tool used in this research was a circular saw blade CMT ORANGE TOOLS multi-rip with rakers (Pesaro, Italy) with the following characteristics: a diameter (*D*) of 300 mm, an inner diameter (*d*) of 70 mm, a width of 2.2 mm, a cutting width (*W*) of 3.2 mm, the number of saw teeth (*z*) equalling 24, an ATB tooth shape, a hook angle ( $\alpha$ ) of 18°, and a grind angle ( $\beta$ ) of 10°. The blades were made of a hard metal cemented carbide (HV10), the body was made from constructive steel, and the tool had slots on the body to reducing noise and vibrations. The primary purpose of this saw is for rip cuts, where the rakers prevent wood contact with the steel plate and is applicable for wet and dry softwood and hardwood.



Fig. 1. Working tool CMT Orange tools

The override values, as the distance from the top point of the tool and the upper surface of the working piece, were measured as 10 mm, 20 mm, and 30 mm, which influenced the incidence angle ( $\varphi$ ). Testing was performed with a constant number of rotations per minute (RPM) of the working spindle (3750 RPM, *i.e.*, at a constant cutting

speed ( $v_c$ ) of 58.875 m/s), whilst the values of the feed rate were 8 and 12 m/min, as suggested by other authors (Souza *et al.* 2011; Nasir and Cool 2019).

The method for determining the cutting power was based on the indirect measurement of the engaged power of the drive electromotor *via* means of a measurement-acquisition portable device (SRD1, Unolux, Belgrade, Serbia) with a sampling frequency of 1 kHz, which was used for data measurement, acquisition, analysis, and processing. The primary purposes of the SRD1 were the measuring, monitoring, processing, and analysing of the data correlated to the power consumption during different types of wood machining. The scale range of the measuring equipment could be set to 5, 10, and 15 kW, according to expected values of engaged power in order to achieve better resolution of the obtained results. The whole system was based upon Power Expert 2.0 software platform (Mandić *et al.* 2015).

The measurement of the surface roughness was performed with a stylus contact tester (TimeSurf TR200, Beijing TIME High Technology Ltd., Beijing, China) in accordance with ISO standard 4287 (1997). The measured roughness parameter was the  $R_a$ , which is the arithmetic average value of the filtered roughness profile determined from deviations about the centre line within the evaluation length. During data acquisition, the reference length was set at 2.5 mm (in accordance with the recommendation of ISO standard 4288 (1996)), whilst the diameter of the diamond stylus tip was 2  $\mu$ m, and the stylus was pressed on the surface with a force of 4 mN. The measurements were made perpendicular to the direction of the wood grain.

The acoustic emission data were sampled using OscilloMeter 7.31 software with a recording frequency range between 0 and 20 kHz. It was possible to obtain both 2D and 3D time dependant spectra, which were analysed latter. The experimental setup used in this research was the same as in a previous paper (Mandić *et al.* 2015).

The distance from the saw blade to the microphone was 1200 mm away from the operating tool in the horizontal direction and 950 mm in the vertical direction from the ground, aligned with the machine work surface (Tanaka 1988; Iskra and Tanaka 2006). The sound produced was undamped, as suggested by Cheng *et al.* (1998). The microphone was a SHURE Beta 18 (Niles, IL). In order to simplify the presented results, the different cutting regimes will be denoted as follows: the feed rate ( $V_{F1}$ ) is 8 m/min and  $V_{F2}$  is 12 m/min; the tool override ( $h_1$ ) is 10 mm,  $h_2$  is 20 mm, and  $h_3$  is 30 mm. The number of cuts for each plank and each cutting regime was 5 (6 groups of specimens with 5 repeats). The number of replicates for every step of the experiment (as shown in Fig.2) was 20, except in the case of *Ra* where it was 40.



Fig. 2. Schematic illustration of experimental steps

### **RESULTS AND DISCUSSION**

Simultaneously with recording the acoustic emissions, the data for the cutting power (P) were recorded. The results for the obtained average power consumption values are presented in Table 1.

No.	V <sub>F</sub> (m/min)	<i>h</i> (mm)	<i>P</i> (W)	No.	V <sub>F</sub> (m/min)	<i>h</i> (mm)	<i>P</i> (W)
1.	8	10	740.15	16.	12	10	1033.88
2.	8	10	727.24	17.	12	10	1042.03
3.	8	10	707.43	18.	12	10	1003.18
4.	8	10	686.19	19.	12	10	984.72
5.	8	10	678.5	20.	12	10	945.1
6.	8	20	747.28	21.	12	20	928.28
7.	8	20	748.72	22.	12	20	900.63
8.	8	20	782.59	23.	12	20	888.16
9.	8	20	734.84	24.	12	20	986.1
10.	8	20	749.45	25.	12	20	984.37
11.	8	30	787.97	26.	12	30	1012.18
12.	8	30	834.15	27.	12	30	972.32
13.	8	30	824.59	28.	12	30	927.35
14.	8	30	782.65	29.	12	30	914.65
15.	8	30	750.11	30.	12	30	963.63

 Table 1. Average Values of Power Consumption for Different Cutting Regimes

Analysis of the obtained data, *i.e.*, the power consumption during circular saw cutting of beech wood, is presented below. A univariate variance analysis ANOVA was applied for testing the significant dependence of the cutting regimes on power consumption.

	SS	Degrees of Freedom	MS	F	р				
Intercept	22133750	1	22133750	18177.17	0.000000				
V <sub>F</sub> (m/min)	342341	1	342341	281.14	0.000000				
<i>h</i> (mm)	5347	2	2673	2.20	0.133162				
V <sub>F</sub> *h	24789	2	12395	10.18	0.000629				
Error	29224	24	1218	-	-				
* Significant re	* Significant results are highlighted in red								

**Table 2.** Univariate Significance Test for *P* with Respects to the  $V_F$  and *h* 

With a probability threshold of 95%, it is possible to say that the feed rate significantly influenced the level of power consumption in a positive manner. At the same time, the override was not enough to influence the measured power consumption (Table 2). However, the mutual impact of the feed rate and override was shown to have a significant effect upon the power consumption (a *p*-value of 0.000629 was less than 0.05).

PEER-REVIEWED ARTICLE

According to the results presented in Table 2, the effect of the feed rate (set at a value of 12 m/min) on the power consumption was the highest when the tool override was 10 mm, which was a possible influence of the average number of incidence teeth (Kovač and Mikleš 2010; Kvietkovà *et al.* 2015).

	VF	h	{1}	{2}	{3}	{4}	{5}	{6}	
	(m/min)	(mm)	707.90	752.58	795.89	1001.8	937.51	958.03	
1	8	10	-	0.358633	0.006479	0.000138	0.000138	0.000138	
2	8	20	0.358633	-	0.391307	0.000138	0.000138	0.000138	
3	8	30	0.006479	0.391307	-	0.000138	0.000149	0.000139	
4	12	10	0.000138	0.000138	0.000138	-	0.073151	0.380590	
5	12	20	0.000138	0.000138	0.000149	0.073151	-	0.934841	
6	12	30	0.000138	0.000138	0.000139	0.380590	0.934841	-	
* Signi	* Significant results are highlighted in red								

Table 3. Tukey HSD Test Results

As shown in Table 3, it became obvious that there was no significant influence of the feed rate ( $V_F$  of 8 m/min) for gradual changes in the h, from 10 mm to 20 mm, and from 20 mm to 30 mm. However, it was clear that considerable changes in power consumption could be expected when the h at 10 mm was increased to 30 mm. Altering the values of h had no significant importance when the value of  $V_F$  was set to 12 m/min (as shown in Fig. 3).



Fig. 3. Mutual impact of the feed rate and tool override on the power consumption

The results of the average values for the  $R_a$  of different cutting regimes are presented in Table 4.

No	. V <sub>F</sub> (m/min)	<i>h</i> (mm)	R <sub>a</sub> (μm)	No.	V <sub>F</sub> (m/min)	<i>h</i> (mm)	R <sub>a</sub> (μm)
1.	8	10	5.4279	16.	12	10	7.0825
2.	8	10	6.8762	17.	12	10	6.4053
3.	8	10	8.9955	18.	12	10	5.9295
4.	8	10	5.9027	19.	12	10	8.2352
5.	8	10	5.1291	20.	12	10	7.382
6.	8	20	5.8101	21.	12	20	8.3066
7.	8	20	6.2847	22.	12	20	7.7285
8.	8	20	7.3951	23.	12	20	5.777
9.	8	20	7.0389	24.	12	20	8.5468
10	. 8	20	6.1895	25.	12	20	10.0569
11	. 8	30	8.1567	26.	12	30	4.8967
12	. 8	30	6.5133	27.	12	30	8.5618
13	. 8	30	7.6998	28.	12	30	8.7259
14	. 8	30	8.6662	29.	12	30	9.8191
15	. 8	30	8.9519	30.	12	30	10.0601

|--|



Fig. 4. Mutual impact of the feed rate and tool override on the Ra

At a significance threshold of 5% and a probability level of p = 0.95 it was found that the  $V_F$  had a significant influence upon the surface roughness, whilst there was no notable impact of the *h* values on the surface roughness (Table 5).

	SS	Degrees of Freedom	MS	F	р
Intercept	13700.90	1	13700.90	4545.057	0.000000
V <sub>F</sub> (m/min)	29.44	1	29.44	9.767	0.001955
<i>h</i> (mm)	8.22	2	4.11	1.364	0.257194
V⊦*h	6.66	2	3.33	1.105	0.332664
Error	886.25	294	3.01	-	-
* Significant	results are high	liahted in red			

It is also possible that the combined effect of the  $V_F$  and h did not cause a significant change in the  $R_a$ . As a matter of impact nature, it could be said that the increased  $V_F$  increased the  $R_a$  (Table 3).

Results obtained during the acoustic emission recordings are presented as 2D and 3D spectra. Examples of the investigated spectra are given in Figs. 5 and Fig. 6.



**Fig. 5.** Acoustic pressure spectra for idling machine in: (a) 2D, frequency on the x-axis and AP on the y-axis; and (b) 3D, time axis added



**Fig. 6.** Acoustic pressure spectra for u2 and h1 in: (a) 2D, frequency on the x-axis and AP on the y-axis; and (b) 3D, time axis added

There was obvious difference between the AP spectra of the idling and operating machine. The solo AP peak at 1.8 kHz on the spectra of the idling machine did not appear on other records related to the operating machine. The entire spectral area of the operating machine spectra was elevated compared to the idling machine spectrum, with some constantly repeating peaks. The AP peaks of particular interest were at 1.6 kHz, 3 kHz, 9 kHz, and 15 kHz whilst investigating a frequency area ranging from 200 Hz to 1.5 kHz on the spectra of the operating machine. The results of the AP obtained from the selected frequency areas are presented in Table 6.

Freque	ency		200 Hz to 1.5 kHz				1.6 kHz				
V <sub>F</sub> (m/min)	<i>h</i> (mm)		AP (dB)						AP (dB	)	
8	10	-70,5	-69,6	-66,2	-69,3	-68,9	-35	-36	-33	-38	-35
8	20	-65	-73	-64	-62	-65	-33	-41	-31	-35	-33
8	30	-63,5	-56	-55	-59	-58	-29	-31	-39	-33	-32
12	10	-61	-63	-65	-68	-67	-25	-31	-32	-33,3	-28
12	20	-58	-57	-59	-56	-61	-28	-28	-32	-25	-31
12	30	-55	-56	-53	-57	-55	-29	-31	-22	-27	-30
Freque	ency			3 kHz					4.5 kHz	z	
V <sub>F</sub> (m/min)	<i>h</i> (mm)		AP (dB) AP (dB)				)				
8	10	-35	-39	-35	-33	-36	-41	-42	-40	-39	-42
8	20	-35	-39	-33	-30	-32	-31	-30	-32	-36	-31
8	30	-33	-37	-36	-31	-33	-30	-29	-30	-31	-31
12	10	-32	-40	-50	-52	-37	-31	-38	-32	-40	-40
12	20	-25	-28	-32	-31	-31	-32	-33	-29	-28	-27
12	30	-30	-29	-21	-30	-31	-29	-28	-29	-29	-30
Freque	ency			9 kHz					15 kHz	2	
V <sub>F</sub> (m/min)	<i>h</i> (mm)			AP (dB)	)				AP (dB	)	
8	10	-100	-100	-100	-96	-100	-110	-110	-110	-108	-107
8	20	-88	-84	-91	-86	-85	-100	-96	-103	-97	-95
8	30	-88	-90	-89	-90	-92	-99	-100	-100	-100	-103
12	10	-89	-96	-90	-100	-110	-101	-108	-103	-110	-115
12	20	-94	-92	-98	-90	-100	-110	-120	-121	-123	-116
12	30	-94	-91	-86	-85	-87	-108	-105	-99	-98	-100

# **Table 6.** Results of the Acoustic Pressure for Different Feed Rate and Tool Override Values at Characteristic Frequencies

According to the presented results in Table 7, it is possible that both the feed rate and tool override had a significant influence on the AP values at frequency areas ranging from 200 to 1.5 kHz, with confidence of 95%. The statistical analysis of the mean values of the areas observed, demonstrated an increase in AP as both the  $V_F$  and h increased.

**Table 7.** Univariate Significance Test for the AP (dB) with Respects to the  $V_F$  and *h* at a Frequency Range of 200 Hz to 1.5 kHz

	SS	Degrees of Freedom	MS	F	р			
Intercept	114824.5	1	114824.5	15183.41	0.000000			
V <sub>F</sub> (m/min)	182.5	1	182.5	24.14	0.000052			
<i>h</i> (mm)	510.3	2	255.2	33.74	0.000000			
V <sub>F</sub> *h	27.9	2	14	1.85	0.179632			
Error	181.5	24	7.6	-	-			
* Significant re	* Significant results are highlighted in red							

Results for the significance of the influence of the cutting parameters on the values of the AP peak at 1.6 kHz are presented in Table 8. According to the results, a significant influence was only found for the  $V_{F}$ . The significance was determined with a probability of 95%.

**Table 8.** Univariate Significance Test for the AP (dB) with Respects to the  $V_F$  and *h* at a Frequency of 1.6 kHz

	SS	Degrees of Freedom	MS	F	р
Intercept	29846.30	1	29846.30	2806.312	0.000000
V <sub>F</sub> (m/min)	222.77	1	222.77	20.946	0.000122
<i>h</i> (mm)	27.40	2	13.70	1.288	0.294120
V <sub>F</sub> *h	0.84	2	0.42	0.039	0.961454
Error	255.25	24	10.64	-	-
* Significant re	sults are hig	hliahted in red			

The AP values obtained from the spectral peak at 3 kHz for different cutting regimes indicated that *h* alone had a significant influence, and the combination of *h* and  $V_F$  also had a significant influence, with probability of 95%. However,  $V_F$  alone had no significant influence on the AP. This statement is confirmed with the results shown in Table 9.

**Table 9.** Univariate Significance Test for the AP (dB) with Respects to the  $V_F$  and *h* at a Frequency of 3 kHz

	SS	Degrees of Freedom	MS	F	р
Intercept	34408.53	1	34408.53	1710.449	0.000000
V <sub>F</sub> (m/min)	10.80	1	10.80	0.537	0.470830
<i>h</i> (mm)	381.27	2	190.63	9.476	0.000926
V <sub>F</sub> *h	230.60	2	115.30	5.732	0.009230
Error	482.80	24	20.12	-	_
* Significant re	sults are high	lighted in red			

	V <sub>F</sub> (m/min)	<i>h</i> (mm)	{1} -35.60	{2} -33.80	{3} -34.00	{4} -42.20	{5} -29.40	{6} -28.20
1	8	10	-	0.987208	0.992528	0.222509	0.280631	0.133757
2	8	20	0.987208	-	1.000000	0.066083	0.636459	0.385154
3	8	30	0.992528	1.000000	-	0.076485	0.593098	0.348122
4	12	10	0.222509	0.066083	0.076485	-	0.001878	0.000745
5	12	20	0.280631	0.636459	0.593098	0.001878	-	0.998103
6	12	30	0.133757	0.385154	0.348122	0.000745	0.998103	-
* Signi	ificant resu	ilts are h	ighlighted in	red				

**Table 10.** Tukey HSD Test Results for the Mutual Effect of  $V_F$  and h on the AP at 3 kHz

The nature of the influence of h on the AP at 3 kHz indicated an increase in AP as the h increased. This increase in the AP values was higher when the h shifted from 10 mm to 20 mm than when it shifted from 20 mm to 30 mm. The mutual effect of different cutting regimes is illustrated by the results of the Tukey test (as presented in Table 10). It can be said the same effect noticed for the influence of h on the AP at a frequency of 3 kHz was also found in the combined effect of  $V_F$  and h.

The Tukey test verified the previously stated effects and also indicated a greater growth in the AP at 3 kHz when the  $V_F$  was 12 m/min, which also meant that there was a significant mutual effect only when the *h* values were 20 mm and 30 mm.

Another point of interest on the operating machine spectra was at 4.5 kHz. The results of the significance test presented in Table 11 demonstrated the significant influence of both cutting parameters, but not their mutual impact, on the recorded values of the AP.

Table 11. Univariate	Significance -	Test for the	AP (dB) w	ith Respects	to the $V_F$
and h at a Frequency	y of 4.5 kHz				

	SS	Degrees of Freedom	MS	F	р			
Intercept	32670.00	1	32670.00	5714.869	0.000000			
V <sub>F</sub> (m/min)	53.33	1	53.33	9.329	0.005451			
<i>h</i> (mm)	462.20	2	231.10	40.426	0.000000			
V⊦*h	15.27	2	7.63	1.335	0.281936			
Error	137.20	24	5.72	_	-			
* Significant results are highlighted in red								

The effect of the  $V_F$  on the AP at a 4.5 kHz peak suggested the AP increased as the  $V_F$  increased. The effect of the *h* on the AP at a 4.5 kHz peak indicated that an increase in the AP resulted from an increase in the *h*, in the same manner as the effects at a 3 kHz peak.

The significance test of the AP at a 9 kHz peak demonstrated similar behaviour to the 3 kHz peak, which showed the significant influence of the h alone and the mutual effect of both cutting factors (Table 12).

**Table 12.** Univariate Significance Test for the AP (dB) with Respects to the  $V_F$  and *h* at a Frequency of 9 kHz

	SS	Degrees of Freedom	MS	F	р		
Intercept	257798.7	1	257798.7	13152.99	0.000000		
V <sub>F</sub> (m/min)	17.6	1	17.6	0.90	0.352327		
<i>h</i> (mm)	450.2	2	225.1	11.48	0.000317		
V <sub>F</sub> *h	158.1	2	79.0	4.03	0.030919		
Error	470.4	24	19.6	_	-		
* Significant results are highlighted in red							

The effect of the *h* values essentially yielded the same pattern as previously shown, which greatly influenced the AP values, when increase from 10 mm to 20 mm and slowly increasing the AP when further increased. Following the previous statement about the combined influence of the cutting regimes on the AP peak at 9 kHz, with respects to the data shown in Table 13 and Fig. 7, it is possible to say that the effect of the  $V_F$  was stronger when it was equal to 8 m/min.



Fig. 7. Mutual impact of the feed rate and tool override on the AP at a frequency of 9 kHz

Table 13. Tukey HSD	Test Results for t	he Mutual Effect of	$V_F$ and $h$ on the AP a	at
9 kHz				

	VF	h	{1}	{2}	{3}	{4}	{5}	{6}
	(m/min)	(mm)	-99.20	-86.80	-89.80	-97.00	-94.80	-88.60
1	8	10	-	0.002280	0.027958	0.967376	0.624006	0.010426
2	8	20	0.002280	-	0.887891	0.014557	0.081929	0.986440
3	8	30	0.027958	0.887891	-	0.143495	0.492929	0.997981
4	12	10	0.967376	0.014557	0.143495	-	0.967376	0.060915
5	12	20	0.624006	0.081929	0.492929	0.967376	-	0.268063
6	12	30	0.010426	0.986440	0.997981	0.060915	0.268063	-
* Signi	* Significant results are highlighted in red							

According to the Tukey test (Table 13), the significant mutual effect of the cutting parameters was shown for altering values of h when the  $V_F$  equals 8 m/min, which agreed with previous statements.

The analysis of the AP at 15 kHz proved to be interesting, since a significant influence was recognized for the effects of the  $V_F$ , h, and mutual  $V_F*h$  on the AP value, with a probability of 95% (Table 14).

Table 14. Univariate Significance	Test for the AP	' (dB) with Re	spects to the $V_F$
and <i>h</i> at a Frequency of 15 kHz			

	SS	Degrees of Freedom	MS	F	р		
Intercept	336020.8	1	336020.8	22082.42	0.000000		
<i>v</i> ⊧ (m/min)	326.7	1	326.7	21.47	0.000105		
<i>h</i> (mm)	322.1	2	161.0	10.58	0.000507		
v <sub>F</sub> *h	666.2	2	333.1	21.89	0.000004		
Error	365.2	24	15.2	-	_		
* Significant results are highlighted in red							

In the case of the  $V_F$ , it is interesting that the dependence of the AV was different from all other investigated peaks. As the  $V_F$  value increased, the value of the AV at a frequency of 15 kHz decreased. The separate effect of the *h* on the AP at a frequency of 15 kHz did not behave in the same manner as previous frequencies, slowly increasing the AP value from 10 to 20 mm and drastically increasing its value from 20 to 30 mm. The combined effect of the cutting regimes is shown in Fig. 8, which suggested that when the  $V_F$  was 8 m/min, the effect of the h yielded the higher AP values at 15 kHz, especially when the *h* was 20 mm and 30 mm.



Fig. 8. Mutual impact of the feed rate and tool override on the AP at a frequency of 15 kHz

**Table 15.** Tukey HSD Test Results for the Mutual Effect of  $V_F$  and h on the AP at 15 kHz

	VF	h	{1}	{2}	{3}	{4}	{5}	{6}	
	(m/min)	(mm)	-109.0	-98.20	-100.4	-107.4	-118.0	-102.0	
1	8	10	-	0.002569	0.020897	0.985897	0.014385	0.085300	
2	8	20	0.002569	-	0.944868	0.011915	0.000138	0.643098	
3	8	30	0.020897	0.944868	-	0.085300	0.000139	0.985897	
4	12	10	0.985897	0.011915	0.085300	-	0.003108	0.279241	
5	12	20	0.014385	0.000138	0.000139	0.003108	-	0.000147	
6	12	30	0.085300	0.643098	0.985897	0.279241	0.000147	-	
* Signi	* Significant results are highlighted in red								

According to the Tukey significance test of the mutual effects of the cutting regimes, when the  $V_F$  was 8 m/min, there was a significant influence on the AP at a 15 kHz peak, which agreed with previous statements. However, when the  $V_F$  was 12 m/min, the mutual effects were not significantly influential.



Fig. 9. The effect of the feed rate on the AP at observed frequency areas



Fig. 10. The effect of the tool override on the AP at observed frequency areas



Fig. 11. The combined impact of the feed rate and tool override on the AP at observed frequency areas

The impact of the  $V_F$  alone was detected at a frequency peak of 1.6 kHz and incorporated with the *h* effect at frequencies of 200 to 1.5 kHz, 4.5 kHz, and 15 kHz. The mutual effect of both cutting parameters was found at 3 kHz, 9 kHz, and 15 kHz frequency peaks (as shown in Fig. 9 through Fig. 11).



Fig. 12. The AP to the cutting power correlation for observed spectral areas



Fig. 13. The AP to the Ra correlation for observed spectral areas

According to the established significant relationships between the AP areas and the  $V_F$  and h and the impact of the cutting parameters on the P and  $R_a$ , it is possible to recognize a relationship between the AP areas and the P and  $R_a$ . Namely, by observing the behaviour of the AP was dependant on the cutting power and surface roughness, it was possible to determine four spectral areas of particular interest for cutting process monitoring. These areas were as follows: 200 to 1500 Hz, 1.6 kHz, 4.5 kHz, and 15 kHz. As shown in Fig. 12 and Fig. 13, it was found that in the case of spectral areas at 200 Hz to 1500 Hz, 1.6 kHz, and 4.5 kHz, the P and the  $R_a$  increased, as the recorded acoustic pressure increased. For spectral area 200 to 1500 Hz, a steady increase for both P and  $R_a$  was noted for elevated values of the AP, except for the P at the 1000 W point. The same behaviour of the P and the  $R_a$  can be observed for the AP peak at 1.6 kHz. For the 4.5 kHz frequency peaks almost the same performance occurred with the exception for 1000 W of the P and slightly slipping of the trend for  $R_a$  just above 6.5 µm. No clear correlation was observed for the AP peak at 15 kHz, since there was no obvious ascend or descend of the AP values with increased P and  $R_a$  values.

### CONCLUSIONS

- 1. The power consumption analysis showed a strong relationship between the feed rate and the consumed power. In addition, a strong relationship between the power consumption and tool override was found, but only if the latter changed from 10 to 30 mm.
- 2. Surface roughness consideration indicated a significant influence of the feed rate on the increase in  $R_a$ , without the tool override impacting the surface quality.
- 3. The acoustic pressure spectra for an operating machine performed the same alterations for the chosen values of the cutting parameters. For the instances of the AP peaks at frequencies of 1.6 kHz, 4.5 kHz, and 9 kHz, as well as for the analyzed spectral area ranging from 200 Hz to 1.5 kHz, a strong influence of *V<sub>F</sub>* was noticed. The latter, along

with the previous two statements, led to the conclusion that by recording the AP intensities at these frequencies, the cutting process can be monitored as a means to learn about experimental and production issues.

- 4. The effect of the *h* on the AP was noticed for all investigated spectral areas, except at the 1.6 kHz peak. However, the influence of the *h* alone on the *P* and  $R_a$  was not noticed. Considering the mutual influence of the *h* and  $V_F$  on the *P* in terms of acoustic emission, the machining system straining could be monitored for the *h* value increasing from 10 mm to 30 mm.
- 5. According to the effects of the cutting regime on the *P* and  $R_a$ , as well as on the AP throughout the spectra areas of interest: at frequencies of 200 to 1500 Hz, 1.6 kHz, 4.5 kHz, and 15 kHz. For the first three mentioned frequencies, there was an obvious relation between the increased AP and the cutting regime augmentation. However, it is not possible to make the same conclusion for the 15 kHz peak.
- 6. For a selected h, it is possible to track the changes in observed spectral areas in order to determine the machining system load, expressed as power consumption, for different wood materials in respect to altering the values of  $V_F$ .

### ACKNOWLEDGMENTS

This research was realized as a part of the project "Agreement for Funding Scientific Research NIO in 2020" (Registration No. 451-02-68 / 2020/14/2000169) and financed by the Ministry of Education, Science, and Technological Development of the Republic of Serbia.

## **REFERENCES CITED**

- Beljo-Lučić, R., and Goglia, V. (2001). "Some possibilities for reducing circular saw idling noise," *Journal of Wood Science* 47, 389-393. DOI: 10.1007/BF00766791
- Cheng, W., Yokochi, H., and Kimura, S. (1998). "Aerodynamic sound and self-excited vibrations of circular saw with step thickness I: Comparison of dynamic characteristic between the common circular saw and the circular saw with step thickness," *Journal of Wood Science* 44, 177-185. DOI: 10.1007/BF00521960
- Deja, M., and Licow, R. (2020). "A pilot study to assess manufacturing processes using selected point measures of vibroacoustic signals generated on a multitasking machine," *International Journal of Advanced Manufacturing Technology* DOI: 10.1007/s00170-020-06180-2).
- Delio, T., Tlusty, J., and Smith, S. (1992). "Use of audio signals for chatter detection and control," *Journal of Manufacture Science and Engineering* 114(2), 146-157. DOI: 10.1115/1.2899767
- Durcan, F. M., and Burdurlu, E. (2018). "Effects of some machining parameters on noise level in planning of some wood materials," *BioResources* 13(2), 2702-2714. DOI: 10.15376/biores.13.2.2702-2714

- Hattori, N., and Iida, T. (1999). "Idling noise from circular saws made of metal with different damping capacities," *Journal of Wood Science* 45, 392-395. DOI: 10.1007/BF01177911
- Iskra, P., and Tanaka, C. (2005). "The influence of wood fibre direction, feed rate and cutting width on sound intensity during routing," *Holz als Roh-und Werkstoff* 63, 167-172. DOI: 10.1007/s00107-004-0541-7
- Iskra, P., and Tanaka, C. (2006). "A comparison of selected acoustic signal analysis techniques to evaluate wood surface roughness produced during routing," *Wood Science and Technology* 40, 247-259. DOI: 10.1007/s00226-005-0059-0
- ISO 4287 (1997). "Geometrical product specifications (GPS) Surface texture: Profile method Terms, definitions and surface texture parameters," International Organization for Standardization, Geneva, Switzerland.
- ISO 4288 (1996). "Geometrical product specifications (GPS) Surface texture: Profile method Rules and procedures for the assessment of surface texture," International Organization for Standardization, Geneva, Switzerland.
- Kopecký, Z., and Rousek, M. (2012). "Impact of dominant vibrations on noise level of dimension sawblades," *Wood Research* 57(1), 151-160.
- Kovač, J. and Mikleš, M. (2010). "Research on individual parameters for cutting power of woodcutting process by circular saws," *Journal of Forest Science* 56(6), 271-277. DOI: 10.17221/94/2009-JFS
- Kvietková, M., Gaff, M., Gašparík, M., Kminiak, R., and Kriš A. (2015). "Effect of number of saw blade teeth on noise level and wear of blade edges during cutting of wood," *BioResources* 10(1), 1657-1666. DOI: 10.15376/biores.10.1.1657-1666
- Mandic, M., Svrzic, S., and Danon, G. (2015). "The comparative analysis of two methods for power consumption measurement in circular saw cutting of laminated particleboard," *Wood Research* 60(1), 125-136.
- Nagatomi, K., Yoshida, K., Banshoya, K., and Murase, Y. (1993). "Recognition of wood cutting condition through sounds, 1: Effect of tool wear on the generation of sound in cutting parallel to the grain," *Mokuzai Gakkaishi* 39(5), 521-528.
- Nasir, V., and Cool, J. (2019). "Optimal power consumption and surface quality in the circular sawing process of Douglas-fir wood," *European Journal of Wood and Wood Products* 77, 609-617. DOI: 10.1007/s00107-019-01412-z
- Nasir, V., and Cool, J. (2020). "Characterization, optimization, and acoustic emission monitoring of airborne dust emission during wood sawing," *International Journal of Advanced Manufacturing Technology* 109, 2365-2375. DOI: 10.1007/s00170-020-05842-5
- Nasir, V., Cool, J., and Sassani, F. (2019). "Acoustic emission monitoring of sawing process: Artificial intelligence approach for optimal sensory feature selection," *The International Journal of Advanced Manufacturing Technology* 102, 4179-4197. DOI: 10.1007/s00170-019-03526-3
- Orlowski, K., Sandak, J., and Tanaka, C. (2007). "The critical rotational speed of circular saw: Simple measurement method and its practical implementations," *Journal of Wood Science* 53, 388-393. DOI: 10.1007/s10086-006-0873-5
- Porankiewicz, B., Wieczorek, D., Djurkovic, M., Idzikowski, I., and Węgrzyne, Z. (2021). "Modelling cutting forces using the moduli of elasticity in oak peripheral milling," *BioResources* 16(1), 1424-1437. DOI: 10.15376/biores.16.1.1424-1437
- Souza, E. M. d., Silva, J. R. M. d., Lima, J. T., Napoli, A., Raad, T. J., and Gontijo, T. G. (2011). "Specific cutting energy consumption in a circular saw for eucalyptus stands

Svrzic *et al.* (2021). "On acoustic emission analysis," *BioResources* 16(4), 8239-8257. 8256

VM01 and MN463," *CERNE* 17(1), 109-115. DOI: 10.1590/S0104-77602011000100013

- Stakhiev, Y. M. (2003). "Research on circular saw disc problems: Several of results," *Holz als Roh- und Werkstoff* 61, 13-22. DOI: 10.1007/s00107-002-0353-6
- Suetsugu, Y., Ando, K., Hattori, N., and Kitayama, S. (2005). "A tool wear sensor for circular saws using wavelet transform signal processing," *Forest Product Journal* 55(11), 79-84.
- Wilkowski, J., and Górski J. (2011). "Vibro-acoustic signals as a source of information about tool wear during laminated chipboard milling," *Wood Research* 56(1), 57-66.

Article submitted: June 3, 2021; Peer review completed: September 18, 2021; Revised version received and accepted: September 21, 2021; Published: October 25, 2021. DOI: 10.15376/biores.16.4.8239-8257